

Research Article

# Experimental analysis of VCR Di diesel engine using Calophyllum inophyllum Bio Diesel

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## Abstract

Biodiesel in recent days are so extensively studied for the sake of using it as an alternate fuel in diesel engine which constitutes the major role in transportation. In the current paper the Calophyllum Inophyllum biodiesel is prepared by using thermal cracking process. The thermo properties of the biodiesel derived from the above process is tested according to ASTM standards. Experiments were conducted on Variable Compression ratio Engine and Performance parameters were determined like Brake Power, Brake Specific Fuel Consumption, Volumetric and Brake Thermal Efficiency. Later Exhaust Emission parameters like CO, CO<sub>2</sub>, HC and NO<sub>x</sub> were determined for biodiesel-diesel blends of 20%, 40%, 60%, 80% and 100%, by incrementing load by 25% and varying compression ratio from 15.5 to 17.5 at constant speed of 1500 rpm, injection pressure of 200 bar and injection timing of 27°bTDC. The emissions shown by Pure Calophyllum biodiesel and B20 are much better and cleaner in comparison to diesel wherein the Carbon monoxide emission and Hydrocarbon emission reduced by 52% and 49.8%. The Nitrogen Oxides emission rose due to significant raise in exhaust gas temperature by 2.1%. The Brake specific fuel consumption and Brake Mean Effective pressure increased by 24.3% and 52%.

**Keywords:** Fatty Acid Methyl ester, Calophyllum Inophyllum Biodiesel, Variable compression ratio Diesel Engine, Emissions and Performance.

## 1. Introduction

Biodiesel stands for any diesel fuel replacement derivative from renewable biological reserve. More exclusively, biodiesel is distinct as oxygenated, sulfur-free, biodegradable, non-toxic and eco-friendly alternative diesel oil. Chemically, it can be defined as a fuel self-possessed of mono-alkyl esters or methyl esters of long chain fatty acids derived from renewable sources, for instance vegetable oil, animal fat and used cooking oil which is designated as B100 and also they meet the special requirements such as the ASTM and the European standards. The conversion of vegetable oils into biodiesel is best possible way to use in engine. There are many techniques to convert the vegetable oil into biodiesel and reduce its viscosity. There are many seeds of trees from which the oil can be extracted. The present work is done on Calophyllum Inophyllum from the Clusiaceae family. There are various names given in various languages like honne oil, puna oil etc. Many investigations were done on this oil by varying parameters like load, compression ratio, injection timing and pressure on both performance and emission with pure and blends with diesel. Collection of seeds and oil extraction and then proceeded for

biodiesel production of Calophyllum Inophyllum Linn ("honne") Oil and understand its physical chemical properties. with molar ratio 8:1, KOH were 1.2wt%, temperature 65°C, reaction time 90 minutes were used and testing of parameters as per ASTM 6751 standards. The thermo physical properties such as acid value density, Flash point, Calorific value, Fire point and Moisture, shows of calophyllum methyl esters were 0.702,892 gm/cc, 176°C 37.18MJ/Kg, 182°C and 0.01%. The physico-chemical parameters proves that Calophyllum may be possible replacement as a reliable fuel as per ASTM. (Chavan S.B *et al.*,2013)

B.K Venkanna *et al* 2011 conducted test on CI engine for Honne oil and neat diesel to determine performance and emission characteristics at varied Injector opening. Experiments were carried out with different IOP of 200 to 260 bars. It was observed that the BTE increased when the IOP was varied from 200bar to 240bar because of good atomization and better mixing. The variation of EGT was observed highest at 200 bar for H100 but the thermal efficiency was found lowest. Emission characteristics like CO, HC for H100 dropped as the IOP increased and reached to least at 240 bars. Smoke opacity decreased with increase in IOP. NO<sub>x</sub> emission was higher with increased IOP. It was concluded that CO, HC, smoke opacity reduced as the IOP increased. NO<sub>x</sub> was

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increased as IOP increased. Also, the BTE was high and Ignition delay was reduced with increase in IOP.

S V Channapattanaa *et al.* 2015, conducted testing on the Calophyllum Inophyllum linn oil for determining emission and thermal performance on DI CI VCR engine. Tests were conducted on blends of biodiesel and standard diesel as a fuel at compression ratios of 15, 16, 17 and 18. They found that as the CR increases, BTE increases for increased Honne bio diesel blends. But at highest CR, BTE reduced by 8.9% for Honne biodiesel compared to diesel. As CR increased, BSFC decreased for all biodiesel blends. At maximum CR BSFC is found to be least. Also increase in biodiesel content EGT decreased with increase in CR. Emission Characteristics: At high Compression ratio they observed that CO, HC, Smoke emissions reduced because of the better combustion of the fuel at increased CR. The emission of CO<sub>2</sub> initially decreased and then increased as the blend concentration of Methyl ester increased. NO<sub>x</sub> emission rose for all blends of biodiesel and decreased for Diesel. At higher CR, NO<sub>x</sub> emission is high. As the CR value increases the performance observation of Methyl ester is found similar to that of Conventional diesel.

C. Srinidhi *et al* 2014, prepared Honne oil methyl ester and tests on the CI engine for Performance and emission with Honne Methyl ester and blends were compared with diesel. The performance conclusion was as the load increases BSFC decreases for the fuels and the blends. The BSFC of B 100 is increased 24% to that of diesel fuel. The Brake thermal efficiency increases with the increase in load is what is observed. The BMEP of Diesel has increased by 53% on average with diesel. BTE B20 is similar to diesel but there is a 3% reduction in the performance B100. 11% reduction compared to diesel. EGT: 2% increase for B20 and 5% increase for B100. NO<sub>x</sub> emission of B100 increased by 36%. CO<sub>2</sub> emissions for B100 is 14% and B20 3% respectively with respect to mineral diesel. HC emission B20 exhibits a reduction by 14% and B 100 shows highest reduction of HC by 51%. The results of the investigation showed that the Honne oil methyl ester – diesel blends is comparable to the diesel in the aspect of performance.

Murugan K *et al.* 2015 conducted experiments on B20 and diesel by varying loads on a single cylinder four stroke diesel engine by the rope brake dynamometer. Exhaust emissions were measured with an AVL Di-Gas 444 exhaust gas analyzer. Honne oil has high viscosity and low calorific value than diesel so it has higher specific fuel consumption than diesel. The engine's Brake thermal efficiency increased and the emission of CO, HC and NO<sub>x</sub> were reduced by using B20 blend.

Varathan R Karuppasamy K *et al* 2015 selected calophyllum inophyllum (Honne) oil and biodiesel was prepared by catalytic transesterification process. Later Thermo Fluid properties was found for blends B25, B50, B75, B100. Experiments were conducted on Kirloskar single cylinder four stroke at CR value 16.5:1, Injection Pressure 240 bar, timing 23° bTDC. As Brake

power increased the Brake Thermal efficiency increased for all the blends of biodiesel and diesel. They observed maximum BTE of 38.86% for B25 against 45.16% of diesel. The specific fuel consumption was found high for the blends as the calorific value was less than diesel. The minimum BSFC (B100) is 0.065 kg/kWh higher compared with diesel. The blends showed low CO, CO<sub>2</sub> and hydrocarbon emission when it is compared to the diesel. At higher loads it was observed that the HC emission decreased with increase of load. For biodiesel blends, the NO<sub>x</sub> emissions was high than the diesel which might be due to longer duration of higher temperature combustion. Performance, combustion and emission characteristics of 25% blend of Methyl ester and Diesel was found to be better than other blends and it showed a possible alternative for diesel as far as the results are considered.

Ashish G. Bandewar *et al.* 2015 conducted on test on CI engine at constant speed of 1500rpm, injection timing 27° bTDC and Injection pressures 210bar. The value of CR was varied from 14.5 to 17.5 for Biodiesel-Diesel blends mainly, H25, H50, H75. At full load condition, when the compression ratio is varied from 14.5 to 17.5, the biodiesel had highest CO emission at lower CR. At the highest CR value, CO emission is less for diesel when compared to Methyl ester. The NO<sub>x</sub> emission was found to be high for all blends at low CR value. HC emission decreased with increased CR because of complete combustion. It was concluded that the biodiesel blends had almost similar emission Characteristics with no modifications in the VCR engine. When antioxidants were added it increased BP and reduced BSFC slightly. Also in emission results it showed that CIB20 increased NO<sub>x</sub> but decreased HC and CO emission. The reduction percent of NO<sub>x</sub> emission was 1.6–3.6%, but increased both CO and HC emission. It can be concluded that CIB20 blends with antioxidants can be directly used without any modification in diesel engines.

## 2. Preparation process of Biodiesel:

Amongst the various methods that are available for the preparation of Biodiesel, in this work the biodiesel is prepared by the process of thermal cracking of the crude oil. Thermal cracking is the chemical reaction in which the lower molecular weight products are produced from the organic compound. The crude oil usually has long chains hydrocarbon molecules and also higher viscosity. Cracking is one of the processes in which the chains can be broken and the viscosity can be decreased.

For the purpose of preparation of biodiesel, a three necked flask packed with the column and then a condenser is used as a reactor for crude oil. An amount of 200ml oil is filled in the flask and 1 ml of HCl is used as anti foaming agent. As crude oil has the capability to form foams which can be affected by possible impurities present in it. A heating metal is used for

heating the crude oil for a maximum temperature of 250° C. Porcelain bites were introduced into the flask in order to retain the uniform temperature inside the flask. Glass wool is used for the insulation purpose in order to reduce the heat loss to the surrounding(Deshpande DP *et al*, 2015) The condensed vapors were collected as the biodiesel was formed on the top and at the bottom glycerol was tapped. Later the biodiesel is separated and property tests were performed to understand if it can be used as the possible source for diesel engine. Further the biodiesel blends were prepared by volume fraction basis.

**Table 1:** Property Table of Biodiesel

PROPERTY	B100
Density (gm/ml)	0.768
Viscosity mm <sup>2</sup> /sec	4.3
Flash Point °C	52
Fire Point °C	58
Net heating Value MJ/kg	38690

### 3. Experimental Setup

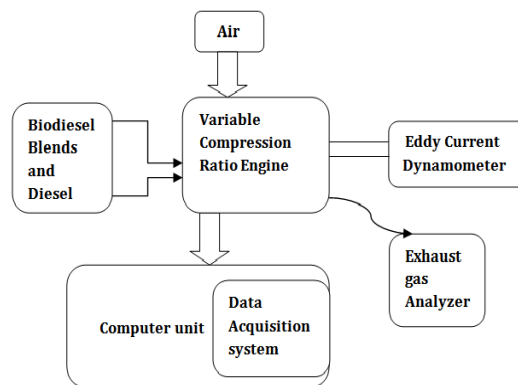
Experimentation was conducted on Diesel Engine test rig having single cylinder four stroke, D.I. water cooled type diesel engine with Eddy current loading for diverse fuel blends. The engine is a variable compression ratio engine.

#### 3.1 Experimental methodology

The testing observations were recorded at constant Engine speed 1500 rpm, injection pressure of 200 bar and injection timing of 27°bTDC. Readings signify the engine parameters recorded were the variation in load and compression ratio with respect to all blends of Diesel-Biodiesel. The measurands are Thermal Performance and Exhaust emissions for different Blends of Bio diesel and the Diesel. The gas analyser used was 5-way in nature that can measure oxides of nitrogen, carbon (like CO and CO<sub>2</sub>) hydrocarbons in PPM or % basis.

**Table 2:** Engine Specification and setup

Make and Model	Rocket Engineering Model VRC -1
Type	4-stroke single cylinder, water Cooled, Variable Compression Ratio from 14.5-17.5
Standard Dimensions of Cylinder	80mm bore diameter and 110mm length of stroke
Speed	1500rpm Constant , governor Controlled
Exhaust Gas Analyser Make	Indus Scientific Pvt.Ltd.
Measureable Gases	CO, CO <sub>2</sub> , NO <sub>x</sub> and HC



**Fig1.** Schematic representation of experimental setup

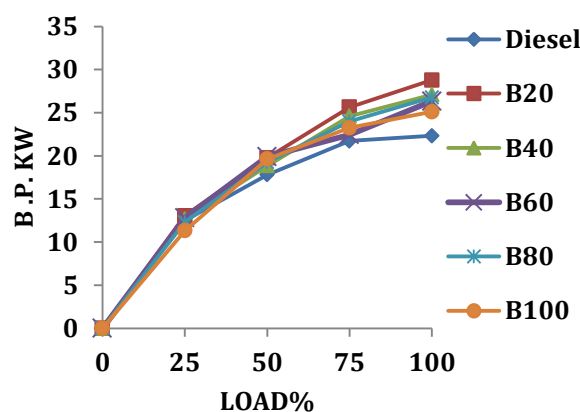
## 4. Results and Discussion

### 4.1 Load Variation: Engine Performance

The below experimental study reveals the variations observed when the load was altered for the blends.

#### 4.1.1 Brake Power

Figure No 2. Illustrates the effect brake power derived from Calophyllum biodiesel to the load that is incremented by 25%. Brake power of an engine is directly proportional to the load applied and hence the BP for all blends shows a linear variation. Also output power is found maximum for B20 Blend which contains 20% vol. Calophyllum methyl ester. The Brake power also derives the knowledge that the actual power delivered or obtained at the crankshaft reminding which is useful observation while considering a alternate fuel for current diesel engines without any high modifications.



**Fig2.** Variation of Brake power to load

#### 4.1.2. Brake Mean Effective Pressure

Brake mean effective pressure is a factor of turbulence occurring due to effective burning inside the cylinder recurring to greater BMEP .The figure 3 signifies that as the load increases the brake mean effective pressure

rises almost linearly. Effective Pressure for diesel is very low in comparison to other mixtures. As the percentage of Calophyllum Methyl ester increases in the blend, relative increase in BMEP is found. Recordings were recorded at engine parameters of Compression ratio 17.5 and the Injection pressure 200bars. The Effective pressure of B100 (Calophyllum oil methyl ester) was maximum than all other the blends of biodiesel-diesel. The BMEP of Diesel has increased by 52.5% on average with conventional fuel.

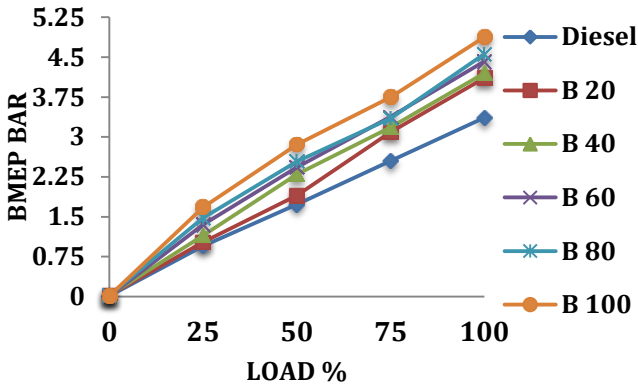


Fig.3. Variation of Brake mean Effective Pressure to load

4.1.3 Exhaust Gas Temperature

The exhaust gas expelled after the Power stroke contains some amount of heat or temperature which is allowed to the atmosphere. The figure 4 plotted below mentioned shows that as the load are altered on the output shaft the expelled gas temperature also increases. The amount of Heat released is directly proportional to exhaust gas temperature and also the heat release is important which is based on burning of fuel or which is also called as complete combustion. The trend observed for all the blends is the same but B20 is increased by 2.4%. The compression ratio 17.5, Injection pressure 200 bar and timing of 27°bTDC were constant during observation. It is inferred from the graph that the B100 pure biodiesel is giving maximum exhaust gas temperature as the load is incremented.

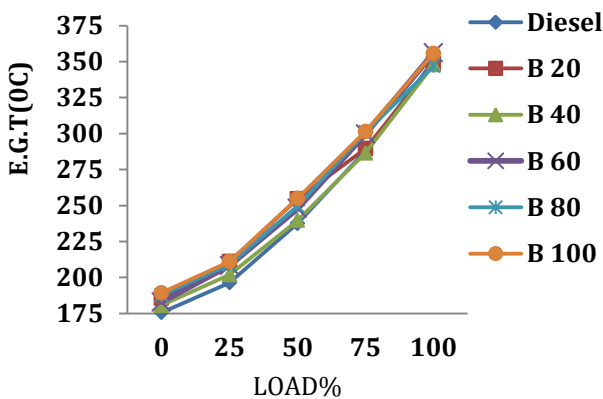


Fig.4. Variation of Exhaust Gas Temperature to load

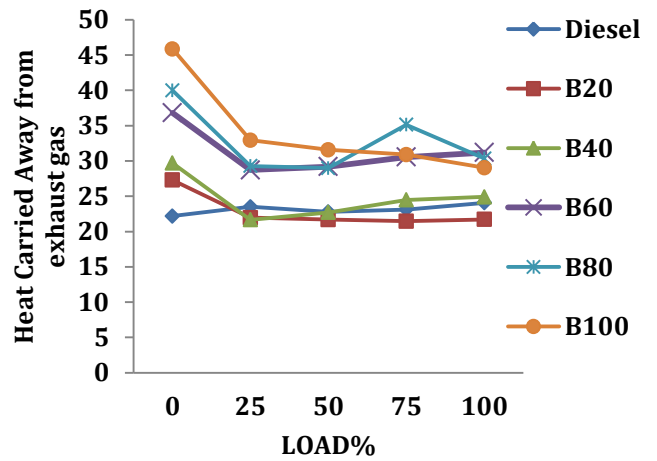


Fig.5. Variation of Heat carried away from exhaust gas to load

4.1.4. Brake Thermal Efficiency

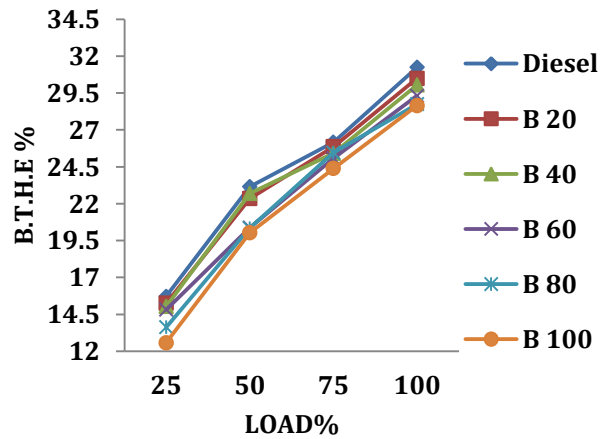


Fig. 6. Variation of Brake Thermal efficiency to load

Brake thermal efficiency (%) is the ratio of output power to heat supplied. The figure no 6 shows that the brake thermal efficiency for diesel is highest as the calorific value of diesel is higher than the other fuels. The brake thermal efficiency of all fuels used in the study is showing an increasing trend to load variation. The heat input provided is given by the mass of fuel consumed and the calorific value of the fuel. The bthe of B20 is almost same at lower loads but as the load increases, the average reduction is 3% with comparison to diesel. In the case of B100 the bthe reduces on a average load of 11%. The brake thermal efficiency of B100 is lowest and fossil diesel is highest.

4.1.5 Brake Specific fuel Consumption

The BSFC shows the amount of fuel consumed for obtaining 1kW of output power. The fig 7 describes the variation of the B.S.F.C with the load in all the biodiesels. The fact that conventional having higher heating value gives least BSFC which is good and as the blend percentage of Callophyllum biodiesel increases

the BSFC Also increases which is disadvantage when such fuels are considered. The BSFC of B20 blends shows the same as diesel and for B100 the Specific fuel consumption is maximum. The bsfc for 100% biodiesel is 24.3% more than that of diesel fuel.

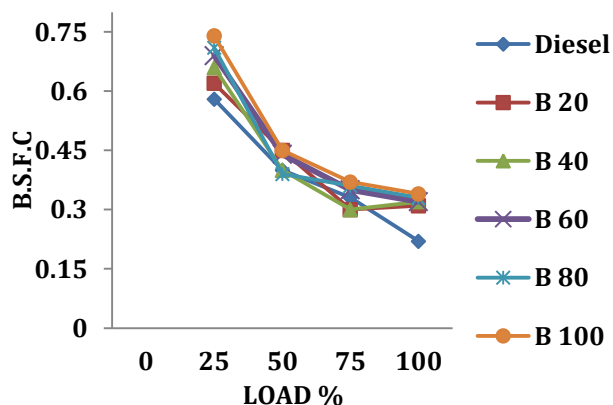


Fig.7. Variation of Brake Specific Fuel to Consumption to load

#### 4.1.6 Volumetric efficiency

The figure 8 indicates the variation of the volumetric efficiency with the load for calophyllum-diesel blends. - The volumetric efficiency is the ratio of actual air taken in to the theoretical air sucked inside the cylinder. The Vol efficiency is highest for B20 blend and lowest for conventional fuel or diesel. For B60 blend it first rises and gradually falls

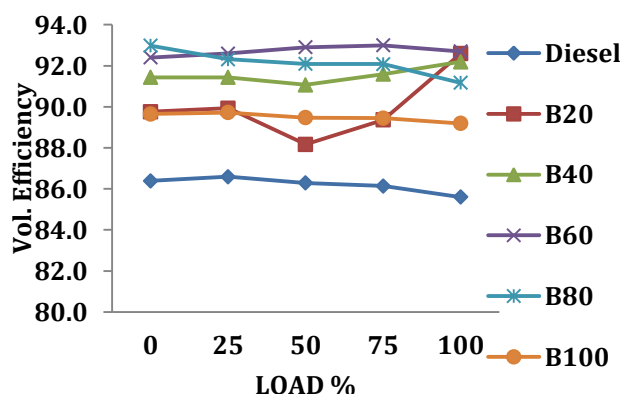


Fig.8. Variation of Volumetric efficiency to load

### 4.2 Exhaust Emission

#### 4.2.1 CO<sub>2</sub> emission

The fig 9 shows the emission of CO<sub>2</sub> with the load incrementation for all the blends. Release of carbon dioxide emission is a sign of cleaner emission. As the amount of the biodiesel percentage increases the CO<sub>2</sub> levels also increases which is good sign for alternate fuel. Even diesel doesn't show the the above statement. When load is increased the fuel consumption

increases, the carbon dioxide emission also shows the result in a linear progressive fashion. The CO<sub>2</sub> emission for B100 is highest and least for diesel. The increase in carbon dioxide emissions for B100 and B20 is 13.5% and 3.8% respectively to diesel.

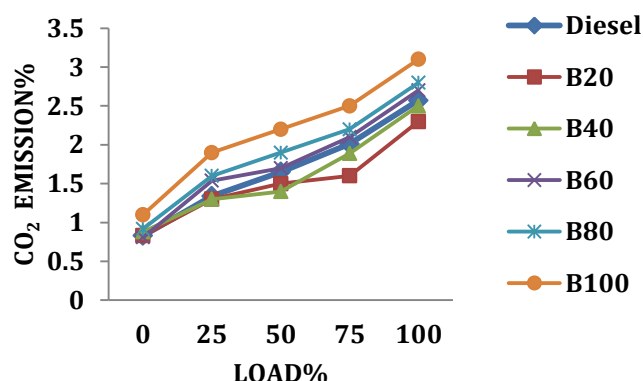


Fig. 9. Variation of CO<sub>2</sub> to load

#### 4.2.2 Carbon-Monoxide Emission

The figure 10 depicts the variation of CO with the increase in load. The load is incremented by 25% i.e. 0, 3, 6, 9, 12 kg's. The CO forms when incomplete burning of fuel takes place. Also flame quenching es CO emission Also these emissions required the following reasons like oxygen content is lesser than theoretical or time shortage for combustion is lesser.

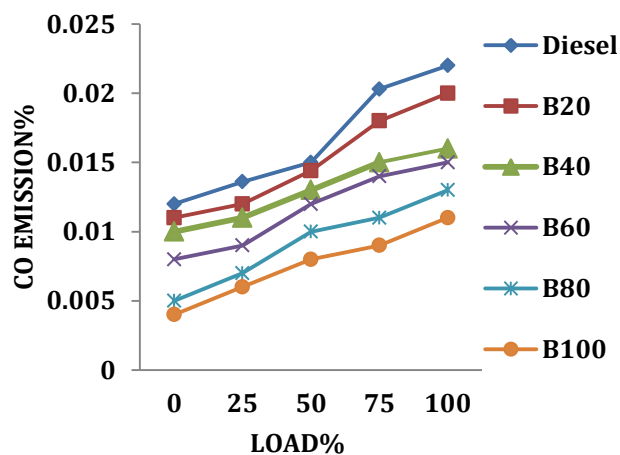


Fig.10 Variation of CO emission to Load.

At constant injection timing of 27°bTDC, the CO emissions of Diesel fuel is highest and is measured in %. The B100 blends show a reduction by 52% than that of diesel and B20 by 12% for all loads.

#### 4.2.3 Hydrocarbon emission

The hydrocarbon emission (HC) is consequential of partial burning. As the load is varied, the HC emission grows. The figure 10 given below indicates the variation in the HC with load in as the blend. The hydrocarbon emission for conventional fuel diesel is

found maximum. As the amount of biodiesel increases in the fuel hydrocarbon emission reduces significantly. B20 shows a reduction of HC emission by 13.5% and B100 by 49.8%.

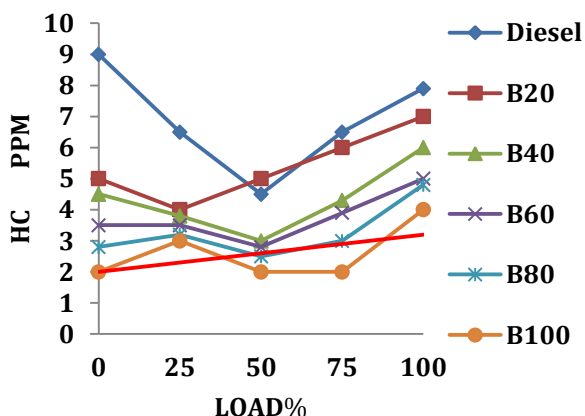


Fig. 11. Variation of Hydrocarbon emission to Load

#### 4.2.4 Nitrogen Oxides emission

The variation of Nitrogen Oxides with respect to load is plotted graphically in figure 11. The nitrogen oxides (NOx) emission is a direct function of burning temperature of the fuel in the combustion chamber inside the engine. The fluctuation of load and Oxides of nitrogen is showing a linear trend. The NOx produced by B100 is highest as combustion is found to be complete as due to the chemical structure of biodiesel has oxygen impregnated in it and which is absent in diesel. The NOx emission of B100 increased by 23% in comparison to diesel

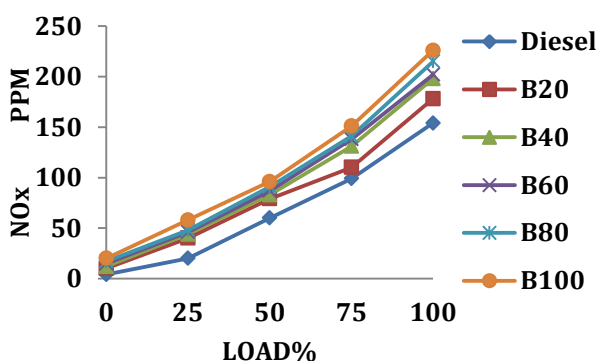


Fig.12 Variation of nitrogen oxide emission to Load

#### 4.3 Compression Ratio Variation: Engine Performance

The below experimental study reveals the variations observed when the Compression ratio was altered for the same blends. The readings were noted at full load or maximum load.

##### 4.3.1 Brake Specific fuel Consumption

It is observed that as the blend percentage increases the brake specific fuel increases which is illustrated in

figure 13. This might be due to, Net heating value of diesel, which is higher than other biodiesel blends. B100 depicts maximum brake specific fuel consumption in comparison to all other blends. The brake specific fuel consumption of B20 blend increases as much as 3.8% in comparison to Diesel fuel for all CR values. The bsfc values of diesel is much lesser in comparison to other blends.

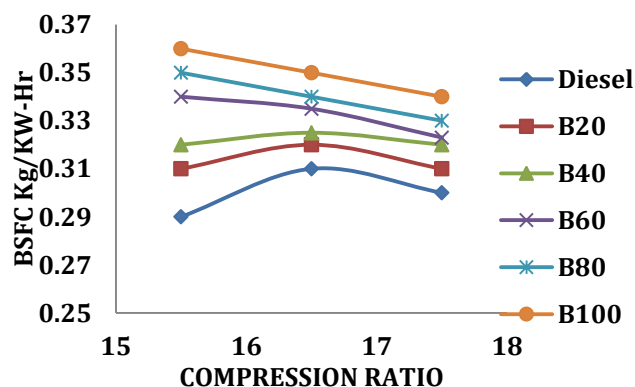


Fig. 13. Variation of BSFC to Compression Ratio

##### 4.3. 2. Brake Thermal Efficiency

The brake thermal efficiency is proportionate to the calorific value of fuel and is usually mentioned in kJ/kg. Also the rate of mass fuel consumed (kg/hr or kg/sec). The Brake thermal efficiency is adverse of Brake specific fuel consumption which is plotted for various compression ratios.

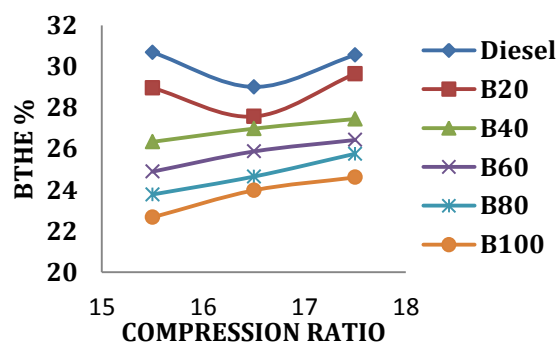


Fig. 14. Variation of BTHE to Compression Ratio

It is observed that the Brake thermal efficiency is highest for conventional diesel and lowest for B 100. Also it is observed that, as the blend percentage increases the Brake thermal efficiency decreases. Even when the CR values are increased the BTHE also increases which is a significance of complete combustion. The appropriate standard compression ratio according to the manufacturer is varied from 17.5 – 18 as depicted in figure 14.

##### 4.3.3 Break Mean Effective Pressure

Brake mean effective pressure is the approximate turbulent pressure developed inside the engine cylinder required to move the piston.

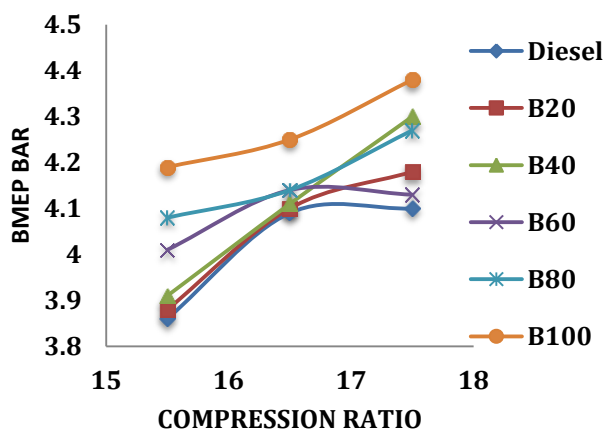


Fig. 15. Variation of BMEP to Compression Ratio

In the figure 15 the Effective pressure increases with CR rating, also the BMEP increase as the blend percentage increases. The Brake mean effective pressure of pure Calophyllum oil is highest as 4.2 bar at compression ratio of 16.5 and 4.45 at CR value 17.5.

#### 4.3.4 Volumetric Efficiency

The Volumetric efficiency of Diesel found is remarkably less than all other fuel blends of bio diesel. The figure 16 shows in the observations that, the percentage of Bio Methyl ester increases the volumetric efficiency at first and later decreases.

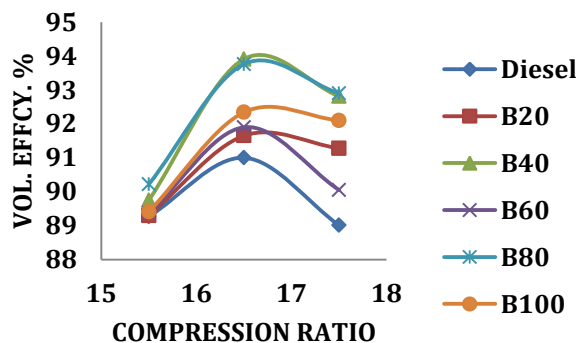


Fig.16.Variation of Vol. Efficiency to Compression Ratio

Also as the percentage of the blend increases, the graph remarks similar characteristics to that of conventional diesel. As the compression ratio increases the volumetric efficiency increases in all the blend and also in the diesel and later decreases. The volumetric efficiency is more for B20 than diesel.

#### 4.4. Exhaust Emission

##### 4.4.1 Carbon dioxide Emission

The figure 17. Shows the graphical representation of Carbon dioxide emission of all the blend and diesel with the increase in compression ratio is shows that carbon dioxide emissions of diesel fuel found is very less and as the blend percentage of biodiesel increases

the carbon dioxide emission also increases which is adverse fact for CO<sub>2</sub> emission. As the compression ratio is increased from 15.5-17.5 in units of 1CR value. The CO<sub>2</sub> emission decreases slightly. The blending plays an important factor showing that, as the blend percentage increases the CO<sub>2</sub> increases. Diesel shows the lowest readings of CO<sub>2</sub>.

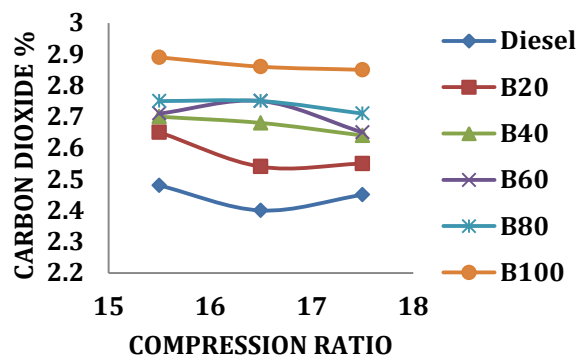


Fig.17.Variation of CO<sub>2</sub> to Compression Ratio

##### 4.4.2 Carbon monoxide Emission

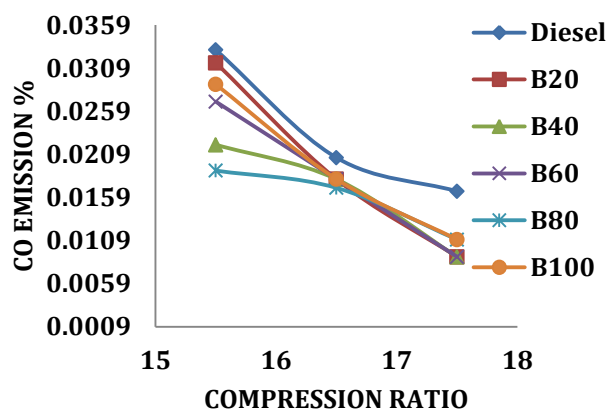


Fig.18.Variation of CO to Compression Ratio

The Figure 18. shows the variation of CO with Compression ratio with all the biodiesel blends. The carbon monoxide emission is a derivative of incomplete combustion, either by insufficient oxygen or time lack to undergo complete combustion. At various compression ratios, the CO emission of diesel fuel is in high. But as the percentage of Bio diesel increases in the Blend, the emission of carbon monoxide decreases. It is found that the lowest CO emission is found for B 100. It might be that oxygen molecule present in the fuel which adds up for combustion thereby increasing the combustion efficiency and effective burning of methyl ester as fuel.

##### 4.4.3. Hydrocarbon Emission

It is observed from the figure 19 that Hydrocarbon emission is inversely proportional to compression ratio. The optimized compression for CI engine when

diesel is used is 17.5 and to this range we may have lesser amount of Hydrocarbon emission. Higher the compression ratio results in cleaner combustion. The hydrocarbon emission is higher in diesel than compared any bio diesel blends .It is seen that B100 has the least HC emission as the compression ratio increased.

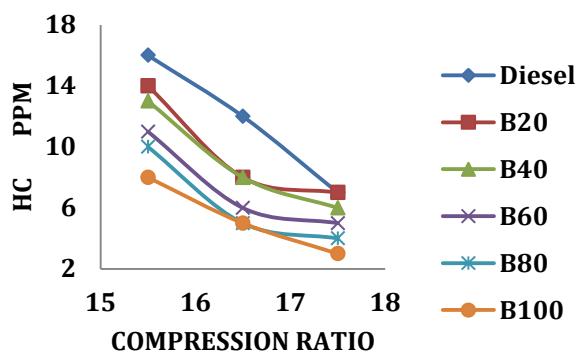


Fig.19. Variation of Hydrocarbon to Compression Ratio.

#### 4.4.5 Nitrogen Oxide Emission

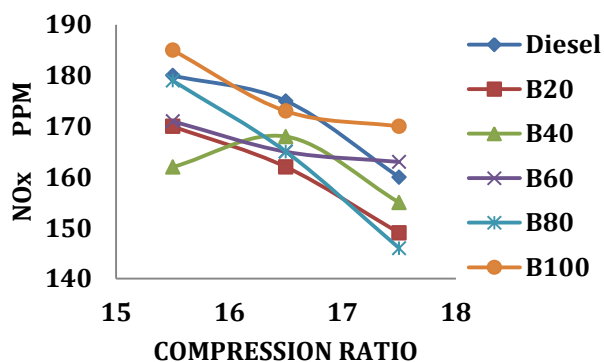


Fig.20. Variation of NOx to Compression Ratio

Formation of Nitrous Oxides is a function of temperature. It is found that the NOx emission is highest for the B100, which might be due to higher Oxygen presence in structure in chemical structure resulting in higher exhaust gas temperature. When the compression ratio increases the emission of NOx decreases for all the blends and also diesel. The NOx emission for B80 and B20 blend is the very less when compared to the other blend .

#### Conclusion

The experimental analysis of biodiesel derived from from calophyllum oil is compared to diesel with the variation of engine load incrementing by 25% and then with the variation of compression ratio observes the following points.

- The Brake thermal efficiency reduced by 3% and 12% for B20 and B100 in comparison of diesel with the variation of load and the brake specific fuel consumption increased by 24.3% for B100 with that of diesel which is a negative point when mileage concept is considered.

- The Brake Mean Effective pressure raised by 52% for B100 which adds to point of higher combustion efficiency.
- The Hydrocarbon and Carbon monoxide emissions reduced by 49.8 % and 52% which proves a cleaner fuel but the NOx increased by 23% for B100 with reference to the load variation.
- While varying compression ratio, the brake thermal efficiency of B20 and B100 reduced by 4% and 21% with comparison to diesel and the Brake specific fuel consumption increased by 14% for B100 as to diesel when the compression ratio is increased.
- The Hydrocarbon emission and the carbon monoxide emission reduced 54% and 17.26% for B100 and 17.1% and 16.5% for B20 with comparison to diesel as compression ratio increased.

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