## Research Article

# A Comparative Study on Shaft Alignment Systems (SAS)

#### Lijesh K.P.<sup>†</sup> and Harish Hirani<sup>‡</sup>

<sup>†</sup>Mechanical Engineering Department, Indian Institute of Technology, Delhi, New Delhi, India

Accepted 02 April 2015, Available online 07 April 2015, Vol.5, No.2 (April 2015)

### Abstract

Improper alignment between the shafts often leads to severe vibration problems in many rotating machines and causes premature failure of components like bearing, coupling, seals etc. To realize the effect of misalignment, an experiment setup was developed and the system performance was evaluated by measuring the system-vibration using accelerometer at different rotational speed (3.5Hz, 7Hz, and 11Hz). The analysis of FFT signals indicated shortcoming of alignment-system based on the mechanical (spirit level + dial gauge) components. The system is then aligned using dual Laser Aligning System (LAS) and experiments were performed again. The results of acceleration signals and corresponding FFT signals have been presented.

Keywords: Alignment, laser aligning system, FFT, Acceleration.

### 1. Introduction

Misalignment in a rotating machinery causes high cost to the industry as it causes premature failure of bearings (Hirani et al, 1998, Hirani et al, 1999, Hirani et al, 2000, Hirani et al, 2000, Hirani et al, 2001, Hirani et al, 2001, Hirani, 2004, Hirani, 2005, Hirani, Suh, 2005, Hirani, Samanta, 2007, Hirani, 2009, Hirani, Verma, 2009, Lijesh, Hirani, 2014, Lijesh, Hirani, 2015, Lijesh, Hirani, 2015, Lijesh, Hirani, 2015, Muzakkir et al, 2011, Muzakkir et al, 2013, Muzakkir et al, 2014, Muzakkir et al, 2015, Rao et al, 2000, Hirani, Samanta, 2007, Shankar et al, 2006) seals (Hirani and Goilkar, 2011, Goilkar and Hirani, 2009, Goilkar and Hirani, 2009, Goilkar and Hirani, 2009, Goilkar and Hirani, 2010), brakes (Sarkar and Hirani, 2015, Sarkar and Hirani, 2013, Sarkar and Hirani, 2013, Sarkar and Hirani, 2013, Sarkar and Hirani, 2013, Sukhwani, Hirani, 2008, Sukhwani et al, 2008, Sukhwani, Hirani, 2008, Sukhwani et al, 2009, Sukhwani et al, 2007), gears (Shah, Hirani, 2014, Hirani 2009), valves (Hirani and Manjunatha, 2007, Muzakkir and Hirani, 2015, Muzakkir and Hirani, 2015), coupling etc.

As per Piotrowski, 1995, 99% of the rotating machinery are misaligned and attaining the perfect alignment between the driving and driven machines is very difficult (Goodwin, 1989, Vance, 1988). Piotrowski, 1995 emphasized that by running any machinery with misalignment reduces life time to less than 1 month from 200 months if the misalignment in the machinery is increased from 0.2mils/inch to 100mils/inch. In other words, the main objective of

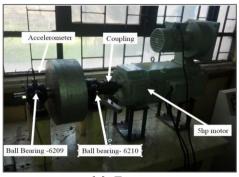
shaft alignment is to increase the operating lifespan of rotating machinery. Therefore, it is necessary to minimize the misalignment in the system.

There are two types of misalignment: parallel and angular misalignments. With parallel misalignment, the center lines of both shafts are parallel but are offset. With angular misalignment, the shafts are at an angle to each other. Alignment error can be parallel misalignment, angular misalignment or a combination of the two. It is expected that every system will have of both parallel combination and angular misalignments. The misalignment can be identified from the FFT plots obtained from the acceleration signals. As per Shekar and Prabhu (1995) the misalignment in the coupling can be identified by observing magnitudes of the frequencies equal to twice the multiples of the rotating frequency.

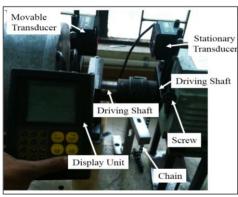
primary industries, mechanical (visual In inspection, dial gauges and spirit levels) aligning techniques are used for aligning the shafts. These aligning methods lack precision and do not provide any assistance to align the shaft-system except intimating the presence of misalignment. There is a need to use more sophisticated aligning system such as Laser Aligning System (LAS). Even though the cost of LAS is comparatively higher, but it may payback in the form of long life of component, energy saving, gain in production. The aim of the present paper is to utilize LAS and compare system performance with the mechanical aligning methods such as combination of spirit level+ dial gauges + visual inspection.

## 2. Experimental Results and Discussion

Experimental setup considered for the present work is shown in figure 1(a). This test setup is consist of a 5hp motor with controller, flywheel of 20kg, and two ball bearings: (i) bearing 1: bearing no.6210 and (ii) bearing 2: bearing no.6209. The jaw type coupling is used to connect the driving shaft (motor) and driven shaft (consisting of ball bearings and flywheel). The accelerometer, mounted on bearing 1 is required to measure the acceleration.



(a) Test setup



(b) LAS attached to the setup

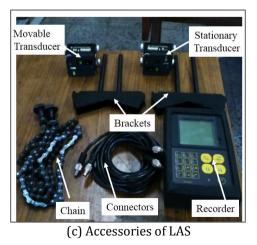


Fig. 1 Experimental setup and LAS

To obtain two rotating shafts co-linear, laser shaft alignment is the fastest and most accurate. The Laser Aligning System (LAS), used in the present work is shown in figure 1(b). The components of LAS are shown in figure 1(c). In LAS, there are two transducer (i) transducer attached to the fixed-system, named as 'stationary transducer' and (ii) transducer attached to the system which has to be adjusted is called movable transducer. In the present case the motor is fixed and system containing driven shaft is to be adjusted.

A sketch of the LAS is shown in figure 2(a). The transducer used in the present work employs two position sensing detectors (PSDs) and laser emitters (as shown in figure 2(a)) to measure the errors in the shaft-alignment. Lasers radiate in a single wavelength, in one direction and in a straight line, and are detected by PSDs. PSDs convert the center of energy of the laser spot into a calibrated digital reading for output to a hand-held display unit.

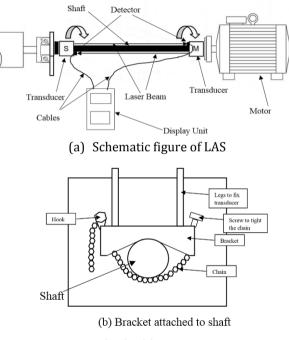


Fig. 2 LAS system

Laser planes are used as references to measure the flatness, straightness or squareness of surfaces or machine axes and at least three reference points in the angular direction are needed to make the laser plane. To obtain the position of the reference point (in the present case it is coupling center), four planes are required (shown in figure 3) (i) 'a'- distance between the stationary transducer (S) and Plane 1, (ii) 'b'distance between the stationary transducer (S) and Plane 2 (M). and (iii) 'c'- distance between the stationary transducer (S) and Plane 3 and (iv) 'd'distance between the stationary transducer (S) and Plane 4. The values of 'a', 'b', 'c' and 'd' needs to be recorded in processing unit of the LAS. The misalignment of the system with respect to the plane 1 from the Plane 3 and plane 4 by using the similar triangle principle shown in figure 4. The actual misalignment (C) with respect to the Plane 3 is calculated by:  $C_{x,y}=h'_{x,y}-(a_{x,y}/c_{x,y})h_{x,y}$ , similarly the procedure is repeated for Plane 4. For parallel alignment two points are enough but for angular adjustment three reading are required.

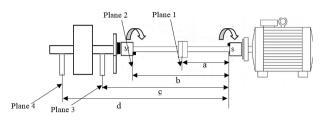
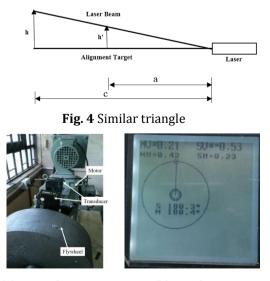


Fig. 3 distance to be measured and fed to LAS

The steps to carry out the aligning of two shafts: (i) driving shaft and (ii) driven shaft are as follows:

- 1. Mount the bracket rail on the motor shaft and flywheel shaft.
- 2. Mount the stationary transducer (Stransducer) and movable transducer (Mtransducer) on the brackets as shown in figure 1(b).
- 3. Connect the S-transducer and M-transducer to the display unit and start the Cardan program to switch on the lasers.
- 4. Adjust the brackets rails so the center (green dots) of the S-transducer is hit by the laser emitted by M- transducer. Similarly, the center (green dots) of the M transducer shall hit by the laser emitted from S-transducer.
- 5. Open the shutter (shown in figure 5(a)). Press Enter button in the display unit to record the first value. The display will indicate a circle symbol when the value is recorded as shown in figure 5(b)
- 6. Turn both the transducers to a degree more than 100 (requirement of LAS) to left side of the motor from the vertical position as shown in figure 6(a) and press Enter button again to record the second value. The display will indicate as shown as shown in figure 6(b).



(a) Transducer position (b) Display unit

Fig. 5 Position of shafts in the vertical position





(a)Transducer position (b) Display unit

Fig. 6 Position of shafts rotated >100 left





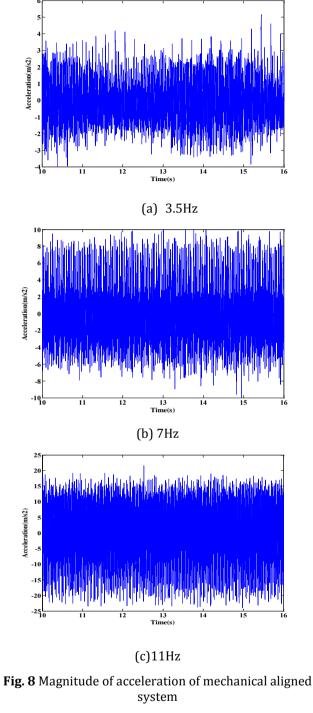
(a)Transducer position (b) Display unit

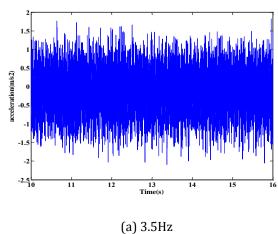
Fig. 7 Position of shafts rotated >100 left

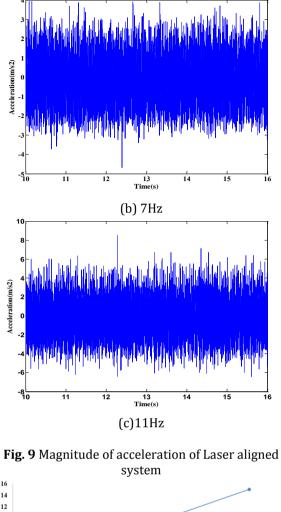
- 7. Turn both the transducers to a degree more than 100 (requirement of LAS) to right side as shown in figure 7(a) and press enter button again to record the second value. The display will indicate as shown as shown in figure 7(b).
- 8. After pressing enter, the deviation in horizontal and vertical direction of two legs will be displayed.
- 9. Based on the obtained values the system is adjusted to obtain '~0' in F1 and F2 in horizontal and vertical directions.

### 3. Results and Discussion

The acceleration signals were acquired using accelerometer at different rotational speed (3.5, 7, and 11Hz). The results for system aligned using mechanical aligning technique and LAS are plotted in figure 8 and 9 respectively. Figures 8(a)-(c) show the acceleration magnitude for 3.5Hz, 7Hz and 11Hz for mechanical aligned system and Figures 9(a)-(c) shows the acceleration magnitude for 3.5Hz, 7Hz and 11Hz for LAS system. The comparison of the magnitude of the accelerations is plotted in figure 10. From this figure it can be concluded that with the increase in rotational speed the magnitude of vibration increases in both the cases but the increase is 2.5, 2.6 and 3 times for mechanical aligned system compared to laser aligned system at 3.5Hz, 7Hz and 11Hz respectively.







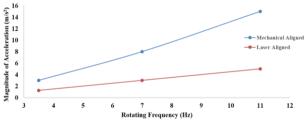
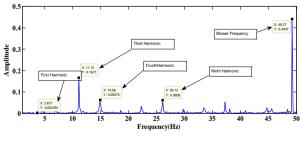
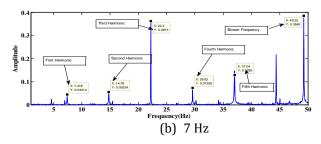


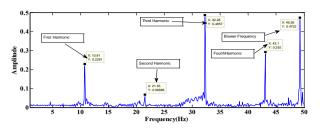
Fig. 10 Magnitude of acceleration for mechanical and laser aligned system



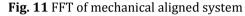




1145| International Journal of Current Engineering and Technology, Vol.5, No.2 (April 2015)



(c) 11 Hz



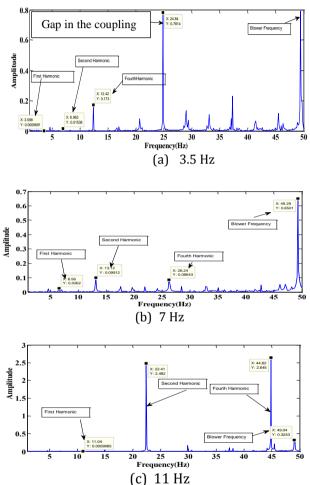


Fig. 12 FFT of Laser aligned system

The FFT for the acquired acceleration signal at different rotational speeds (3.5, 7, and 11Hz) for aligned systems using mechanical method and LAS are plotted in figure 11 and 12 respectively. Figures 11(a)-(c) show the FFT for 3.5Hz, 7Hz and 11Hz for mechanical aligned system. Figures 12(a)-(c) show the FFT for 3.5Hz, 7Hz and 11Hz for system aligned with mechanical unit the dominant frequency set contain both even and odd number of harmonics of the operating frequency. This indicates the existence of misalignment in both coupling and bearings. The peak of 49Hz in the FFT plot, is the operating frequency of the blower which is used for cooling the motor. In the case of the system aligned with LAS, the presence of only the even numbers of

harmonics indicating misalignment only in the coupling. On close investigation of the coupling the deformation of the rubber, used in the jaw type coupling, was observed (as shown in figure 13). This permanent deformation of one side of rubber may have resulted FFT with even number of harmonic signals. Hence it was decided to change the coupling for the future work.



**Fig. 13** Gap between the jaw of the coupling due to permanent deformation of rubber

#### Conclusions

Detailed studies on (i) system using mechanical aligning system and (ii) system using laser aligning system have been carried out. Due to the enormous advantage of Laser aligning system, it was decided to develop an experimental setup and compare its performance with mechanical alignment system. Following conclusion are made:

(i) System aligned by mechanical system resulted in higher vibration compare to system aligned by LAS.

(ii) The system aligned by mechanical system showed both odd and even multiple of peaks indicating misalignment in full system.

(iii) The system aligned by LAS showed even multiple of peaks indicating misalignment in only coupling.

(iv) Visual inspection of coupling indicated permanent deformation of rubber used in lovejoy coupling to absorb the vibration. The the permanent set in rubber, mislaignment occurred in the system. Due to this reason it was decided to change the coupling.

#### Acknowledgment

This research was supported by Council of Scientific and Industrial Research, New Delhi, India [Grant No. 70(0073)/2013/EMR-II].

#### References

- S M Muzakkir, K P Lijesh, H Hirani, and G D Thakre, 2015, Effect of Cylindricity on the Tribological Performance of Heavily-Loaded Slow Speed Journal Bearing, Proc. Institute Mech. Engineers., Part J, Journal of Engineering Tribology, 229(2), 178-195.
- S M Muzakkir, H Hirani and G D Thakre, 2013, Lubricant for Heavily-Loaded Slow Speed Journal Bearing, Tribology Transactions, 56 (6), 2013, 1060-1068.
- S M Muzakkir, H Hirani, G D Thakre and M R Tyagi, 2011, Tribological Failure Analysis of Journal Bearings used in Sugar Mill, Engineering Failure Analysis, 18(8), 2093-2103.
- H. Hirani, K. Athre, S. Biswas, 1999, Dynamic Analysis of Engine Bearings, International Journal of Rotating Machinery. 5(4), 283-293.

- H. Hirani, K. Athre, S. Biswas, 1998, Rapid and Globally Convergent Method for Dynamically Loaded Journal Bearing Design, Proc. IMechE (UK), Journal of Engineering Tribology. 212, 207-214
- H. Hirani, K. Athre, S. Biswas, 2000, A Hybrid Solution Scheme for Performance Evaluation of Crankshaft Bearings, Trans. ASME, Journal of Tribology. 122(4), 733-740
- H. Hirani, K. Athre, S. Biswas, 2000, Transient Trajectory of Journal in Hydrodynamic Bearing, Applied Mechanics and Engineering. 5(2).
- H. Hirani, K. Athre, S. Biswas, 2001, A Simplified Mass Conserving Algorithm for Iournal Bearing under Dynamic Loads. International Journal of Rotating Machinerv. 1. 41-51.
- H. Hirani, K. Athre, S. Biswas, Lubricant 2001, Shear Thinning Analysis of Engine Journal Bearings, STLE, Journal of Tribology Transaction, 44 (1), 125-131.
- H. Hirani, K. Athre, S. Biswas, 2002, Comprehensive Design Methodology for Engine Journal Bearing, IMechE (UK), Part J, Journal of Engineering Tribology. 214, , 401-412.
- H. Hirani, 2004, Multi-objective Optimization of a journal bearing using the Pareto optimal concept, Proc. Institute Mech. Engineers., Part J, Journal of Engineering Tribology. 218 (4), 323-336.
- H. Hirani, Multiobjective optimization of journal bearing using mass conserving and genetic algorithms, Proc. Institute Mech. 2005, Engineers., Part J, Journal of Engineering Tribology. 219(3), 235-248.
- H. Hirani, N.P. Sub, 2005, Journal Bearing Design using Multi-objective Genetic Algorithm and Axiomatic Design Approaches, Tribology International, 38(5), 481-491
- H. Hirani, M. Verma, 2009, Tribological study of elastomeric bearings of marine shaft system, Tribology International, 42(2), 378-390.
- T. V. V. L. N Rao, H. Hirani, K. Athre, S. Biswas, 2000, An Analytical Approach to Evaluate Dynamic Coefficients and Non-linear Transient Analysis of a Hydrodynamic Journal Bearing, STLE Tribology Transactions, 23(1), 109-115.
- S M Muzakkir, K. P. Lijesh, H. Hirani, 2015, Effect of Base Oil on the Anti-Wear Performance of Multi-Walled Carbon Nano-tubes (MWCNT), International Journal of Current Engineering and Technology, 5(2), 681-684.
- H Hirani, T Rao, K Athre and S Biswas, Rapid performance evaluation of journal bearings, Tribology international, 30(11), 1997, 825-834.
- H Hirani, K Athre, S Biswas, 1999, Dynamically loaded finite length journal bearings: analytical method of solution. Journal of tribology, 121(4), 844-852.
- K. P. Lijesh, H. Hirani, Stiffness and Damping Coefficients for Rubber mounted Hybrid Bearing, Lubrication Science, 26(5), 2014, 301-314.
- K. P. Lijesh, H. Hirani, 2015, Development of Analytical Equations for Design and Optimization of Axially Polarized Radial Passive Magnetic Bearing, ASME, Journal of Tribology, 137(1), (9 pages).
- K. P. Lijesh, H. Hirani, 2015, Optimization of Eight Pole Radial Active Magnetic Bearing, ASME, Journal of Tribology, 137(2),
- K.P. Lijesh, H. Hirani, 2015, Design and Development of Halbach Electromagnet for Active Magnetic Bearing, Progress In Electromagnetics Research C, 56, 173–181.
- P. Samanta, H.Hirani, 2007, A Simplified Optimization Approach for Permanent Magnetic Journal Bearing, Indian Journal of Tribology
- P. Samanta, H. Hirani, 2008, Magnetic bearing configurations: Theoretical and experimental studies Magnetics, IEEE Transactions on 44(2), 292-300
- S. Shankar, Sandeep, H. Hirani, Active Magnetic Bearing, Indian Journal of Tribology, 15-25, 2006.
- P. Samanta, H. Hirani, 2007, Hybrid (Hydrodynamic + Permanent Magnetic) Journal Bearings, Proc. Institute Mech. Engineers., Part J. Journal of Engineering Tribology, 881-891. 221(18), K. P. Lijesh, H. Hirani, 2014, Design of eight pole radial active magnetic bearing using monotonicity. In Industrial and Information Systems (ICIIS), 2014 9th International Conference on IEEE, 1-6.
- K. P. Lijesh, H. Hirani, 2015, Magnetic Bearing Using RMD Configuration, Journal of Tribology
- H. Hirani, 2009. Root cause failure analysis of outer ring fracture of four row cylindrical roller bearing, Tribology Transactions, 52(2), 180-190.
- K.P. Lijesh, S. M. Muzakkir, H. Hirani. 2015, Failure Analysis of Rolling Contact Bearing for Flywheel Energy Storage System, 5(1), 439-443.
- H. Hirani, H. S.S.Goilkar, 2011. Rotordynamic Analysis of Carbon Graphite Seals of a Steam Rotary Joint , Book on IUTAM Symposium on Emerging Trends in Rotor Dynamics, Springer Netherlands, 253-262.
- H. Hirani, H. S.S.Goilkar, 2009, Design and development of test setup for online wear monitoring of mechanical face seals using torque sensor, Tribology Transactions, . 52(1), 47-58.
- H. Hirani, H. S.S.Goilkar, 2009, Tribological Characterization of Carbon Graphite Secondary Seal Indian Journal of Tribology, 4(2), 1-6.
- H. Hirani, S.S.Goilkar, 2010, Parametric Study on Balance Ratio of Mechanical Face Seal in Steam Environment, Tribology International, 43(5-6), 1180-1185.
- H. Hirani, S.S.Goilkar, 2009, Formation of Transfer Layer and its Effect on Friction and Wear of Carbon-Graphite Face Seal under Dry, Water and Steam Environments, Wear, 226(11-12), 1141-1154.
- S M Muzakkir, Hirani H, Thakre G D., 2012, Performance evaluation of journal bearings used in sugar mills using Taguchi method. Soc. Tribol. Lubr. Eng. Annu. Meet. Exhib. 2012, p. 121-3.
- Lijesh KP, S M Muzakkir, Hirani H. 2015, Experimental Tribological Performance Evaluation of Nano Lubricant using Multi-Walled Carbon Nano-tubes (MWCNT). Int J Adv Eng Res:1-8.

- S M Muzakkir, Lijesh K P, Hirani H, 2014, Carbon Nanotubes as Solid Lubricant Additives for Anti-wear Performance Enhancement under Mixed Lubrication Conditions. Int J Mod Eng Res;2:1-4.
- Burla RK, Seshu P, Hirani H, Sajanpawar PR, Suresh HS. 2003, Three Dimensional Finite Element Analysis of Crankshaft Torsional Vibrations using Parametric Modeling Techniques
- Gupta S, Hirani H. 2011, Optimization of Magnetorheological Brake. ASME/STLE 2011 Jt. Tribol. Conf., ASME; p. 405–6.
- Hirani H. 2009, Online wear monitoring of spur gears. Soc. Tribol. Lubr. Eng. Annu. Meet. Exhib. 2009, p. 350–2.
- Hirani H. 2009, Magnetorheological smart automotive engine mount. Soc. Tribol. Lubr. Eng. Annu. Meet. Exhib. 2009, p. 353-5.
- Hirani H, Dani S. 2005, Variable valve actuation mechanism magnetorheological fluid. Proc. World Tribol. Congr. III - 2005, p. 569–70.
- Hirani H, Samanta P. 2005, Performance evaluation of magnetohydrodynamic bearing. Proc. World Tribol. Congr. III - 2005, p. 97–8.
- Lijesh KP, Hirani H., 2014, Development of Analytical Equations for Design and Optimization of Axially Polarized Radial Passive Magnetic Bearing. Journal of Tribology American Society of Mechanical Engineers;
- Hirani H, Suh NP. 2005, Journal bearing design using multiobjective genetic algorithm and axiomatic design approaches. Tribol Int;38:481-91.
- Hirani H, Sukhwani VK. Theoretical and experimental studies on magnetorheological brake. Inst. Mech. Eng. - Braking 2009, 2009, p. 75–84. Hirani H, Manjunatha CS. 2007Performance evaluation of a magnetorheological fluid
- variable valve. Proc Inst Mech Eng Part D J Automob Eng; 221:83-93.
- Muzakkir SM, Lijesh KP, Hirani H. 2014, Tribological failure analysis of a heavilyloaded slow speed hybrid journal bearing. Eng Fail Anal;40:97-113.
- Rao TVVLN, Biswas S, Hirani H, Athre K. 2000, An analytical approach to evaluate dynamic coefficients and nonlinear transient analysis of a hydrodynamic journal bearing. Tribol Trans;43:109-15.
- Samanta P, Hirani H. 2009, An overview of passive magnetic bearings. 2008 Proc. STLE/ASME Int. Jt. Tribol. Conf. IJTC 2008, p. 465-7.
- Sukhwani VK, Hirani H. 2008, Design, development, and performance evaluation of high-speed magnetorheological brakes, Proc Inst Mech Eng Part L J Mater Des Appl;222:73-82.
- Sukhwani VK, Hirani H, Singh T, 2007, Synthesis and performance evaluation of magnetorheological (MR) grease, NLGI Spokesm; 71:25
- S M Muzakkir, Lijesh KP, Hirani H. 2014, Tribological failure analysis of a heavilyloaded slow speed hybrid journal bearing. Eng Fail Anal;40:97-113.
- S M Muzakkir, K. P. Lijesh and Harish Hirani, 2015, Effect of Base Oil on the Anti-Wear Performance of Multi-Walled Carbon Nano-tubes (MWCNT), International Journal of Current Engineering and Technology, vol.5, no.2, pp.681-684.
- Samanta F, Hirani H, Mitra A, Kulkarni AM, Femandes BG, 2005, Test setup for magnetohydrodynamic journal bearing, Proc. 12th Natl. Conf. Mach. Mech. NaCoMM 2005, National Conference on Machines and Mechanisms, NaCoMM 2005; p. 298-303.
- S M Muzakkir, & Hirani, H., 2015, Design of Valve System using Optimization Approach, International Journal of Current Engineering and Technology, Vol.5, No.2, pp. 1211-1217.
- S M Muzakkir, 2015, Experimental Investigations on Effect of Tungsten Disulphide (WS<sub>2</sub>) on Wear Performance of Commercial Lubricant Under Mixed Lubrication Conditions, International Journal of Current Engineering and Technology, vol.5, no.2, pp.1112-1114.
- S M Muzakkir, 2015, Enhancement of Wear Performance of Commercial Lubricant Using Zinc (Zn) Nano particles Under Mixed Lubrication Conditions, International Journal of Current Engineering and Technology, vol.5, no.2, pp.1109-1111.
- S M Muzakkir & Hirani, H., 2015, Innovative-Tribo-Design of Variable Valve Timing Mechanism: Need and Concept, International Journal of Current Engineering and Technology, vol.5, no.2, pp.1115-1125.
- S M Muzakkir, 2015, Effectiveness of Carbon Nanotubes (CNT) as Lubricant Additive in Mixed Lubrication Conditions, International Journal of Emerging Trends in Engineering and Development, Issue 5, Vol. 3, pp. 261-266.
- S M Muzakkir, K. P. Lijesh, and Harish Hirani, 2015, Carbon Nanotubes as Solid Lubricant Additives for Anti-wear Performance Enhancement under Mixed Lubrication Conditions, International Journal of Modern Engineering Research, Vol. 5, Iss.2, pp. 1-4.
- Muzakkir SM and Harish H, 2015, Experimental Investigation on Effect of Grinding Direction on Wear Under Heavy Load and Slow Speed Conditions with Molybdenum Disulphide ( $MoS_2$ ) as Additive in Commercial Lubricant, International Journal of Engineering Research, Volume No.4, Issue No.3, pp : 102-
- Muzakkir SM and Harish H, 2014, Experimental Investigation on Effect of Particle Sizes of Molybdenum Disulphide on Wear Under Heavy Load and Slow Speed Conditions, International Journal of Modern Engineering Research, 2014, Vol. 4, Iss. 12, pp. 46-49.
- Muzakkir SM, 2015, Design and Development of Experimental setup for Direct Acting Overhead Camshaft (DOHC) Valve System, International Journal of Current Engineering and Technology, vol.5, no.3, pp.2027-2031.
- Vance JM, (1988), Rotordynamics of Turbomachinery. New York: John Wiley & Sons
- Piotrowski J, (1995), Shaft Alignment Handbook, New York, Marcel Dekker, Inc. Sekhar AS, and B.S.Prabhu, (1995), Effects of coupling misalignment on vibrations of rotating machinery, Journal of Sound and Vibration, Vol.185,

no. 4, pp. 655-671